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(54) **CONE CRUSHER HAVING AN
ARRANGEMENT FOR MEASURING A
POSITION OF A CRUSHING SHELL**

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See application file for complete search history.

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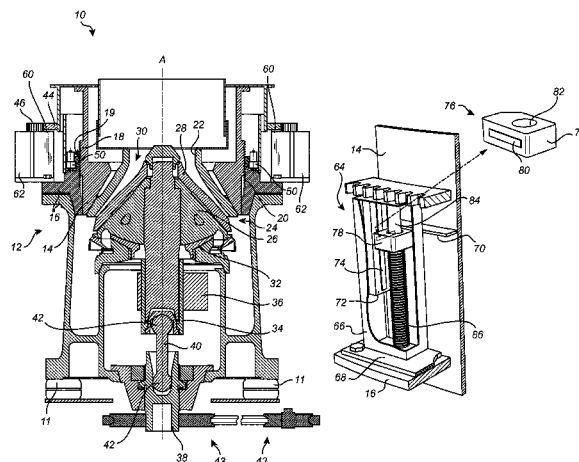
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(57) **ABSTRACT**

A cone crusher includes an outer crushing shell and an inner crushing shell forming a crushing gap there between. The outer crushing shell is supported on an upper frame member in vertically adjustable engagement with a lower frame member. A sensor arrangement having a sensor element is provided for measuring the vertical position of the outer crushing shell. The cone crusher further includes a target means arranged to be detectable by the sensor element, wherein one of the target means and the sensor element are arranged to follow vertical movement of the upper frame member and to move in relation to the other one of the target means and the sensor element. The sensor element includes a vertical sensing array which extends in a vertical direction along at least a portion of a range within which the target means may be moved upon adjusting the vertical position of the upper frame member.

13 Claims, 4 Drawing Sheets



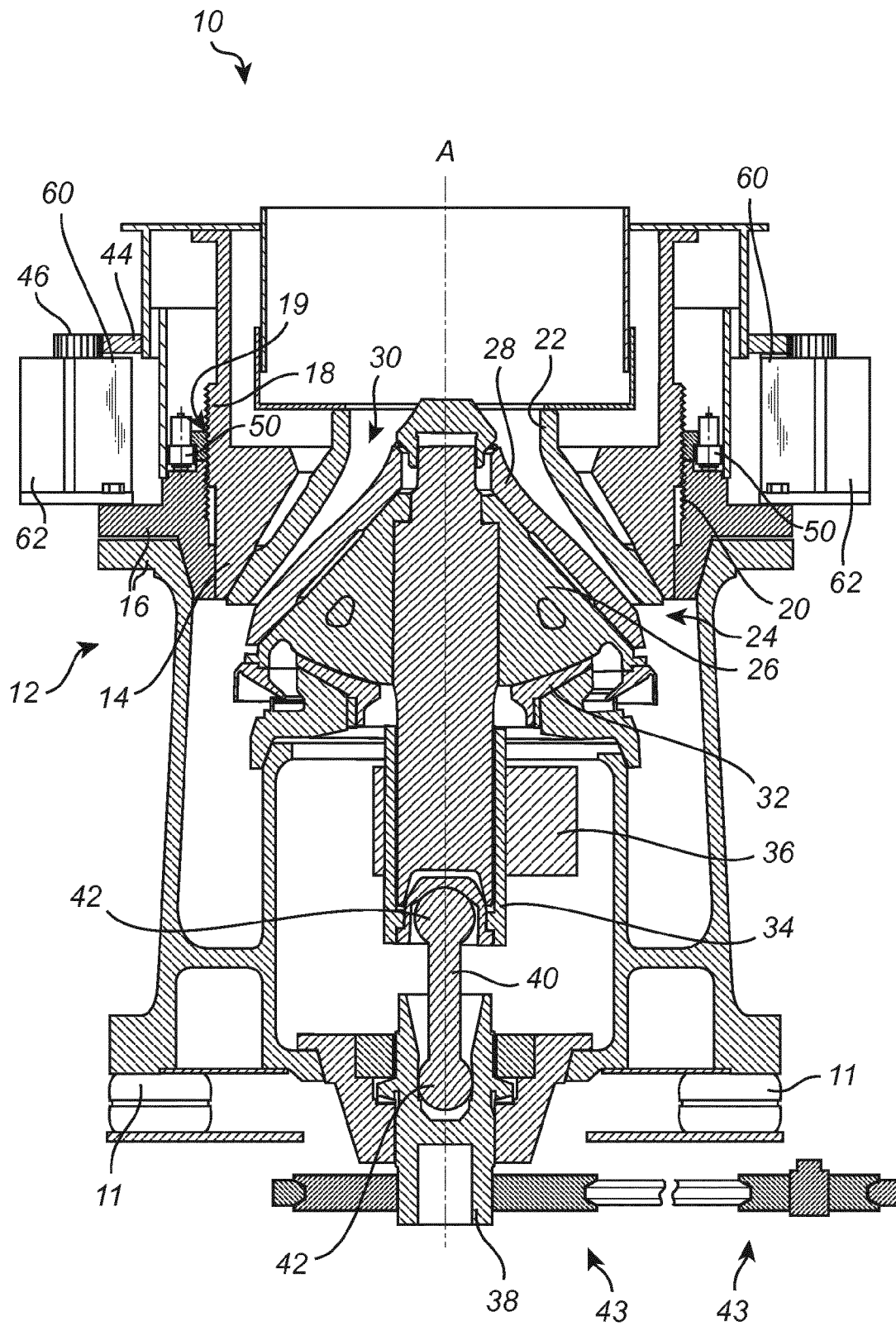


Fig. 1

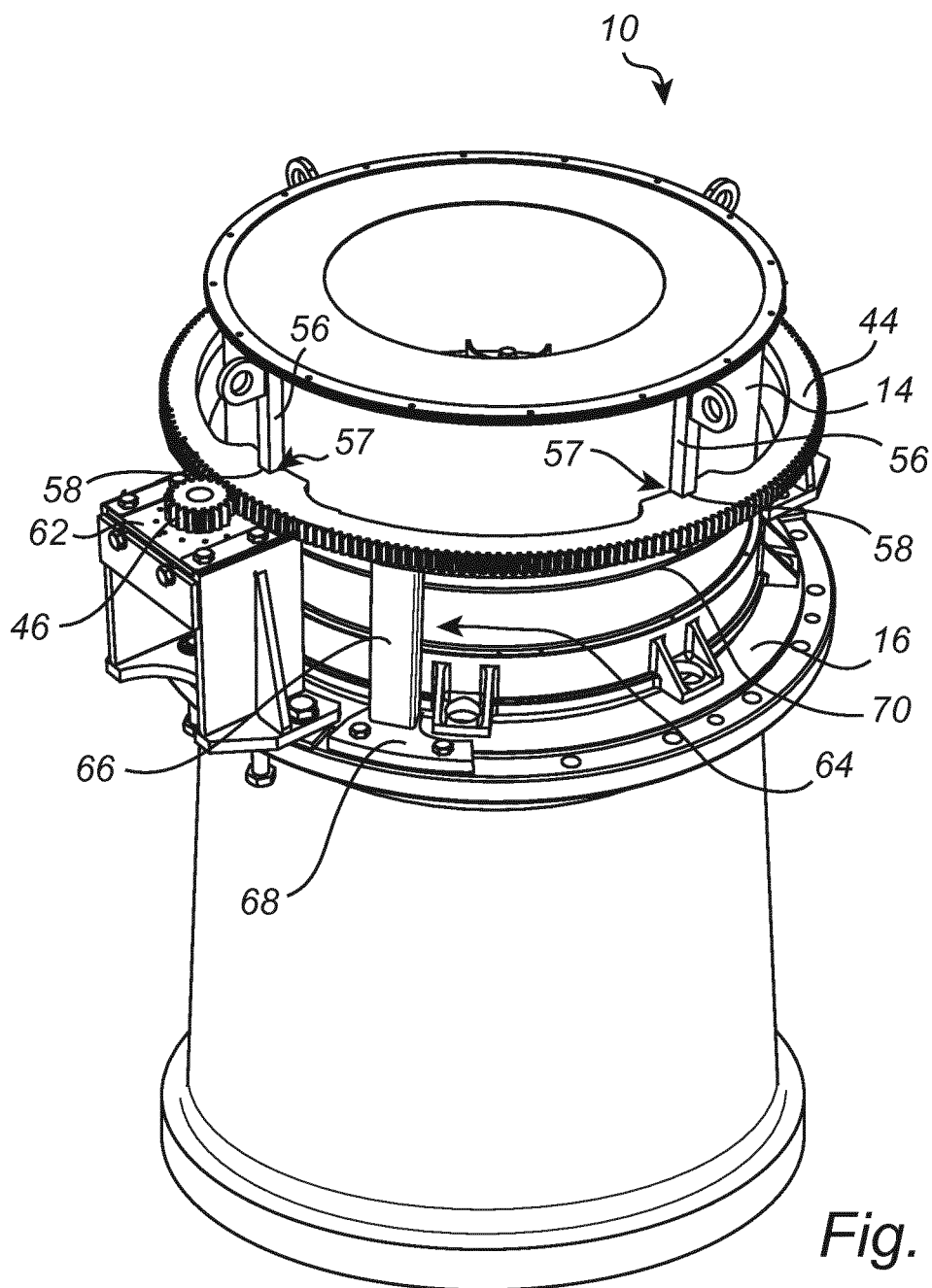


Fig. 2

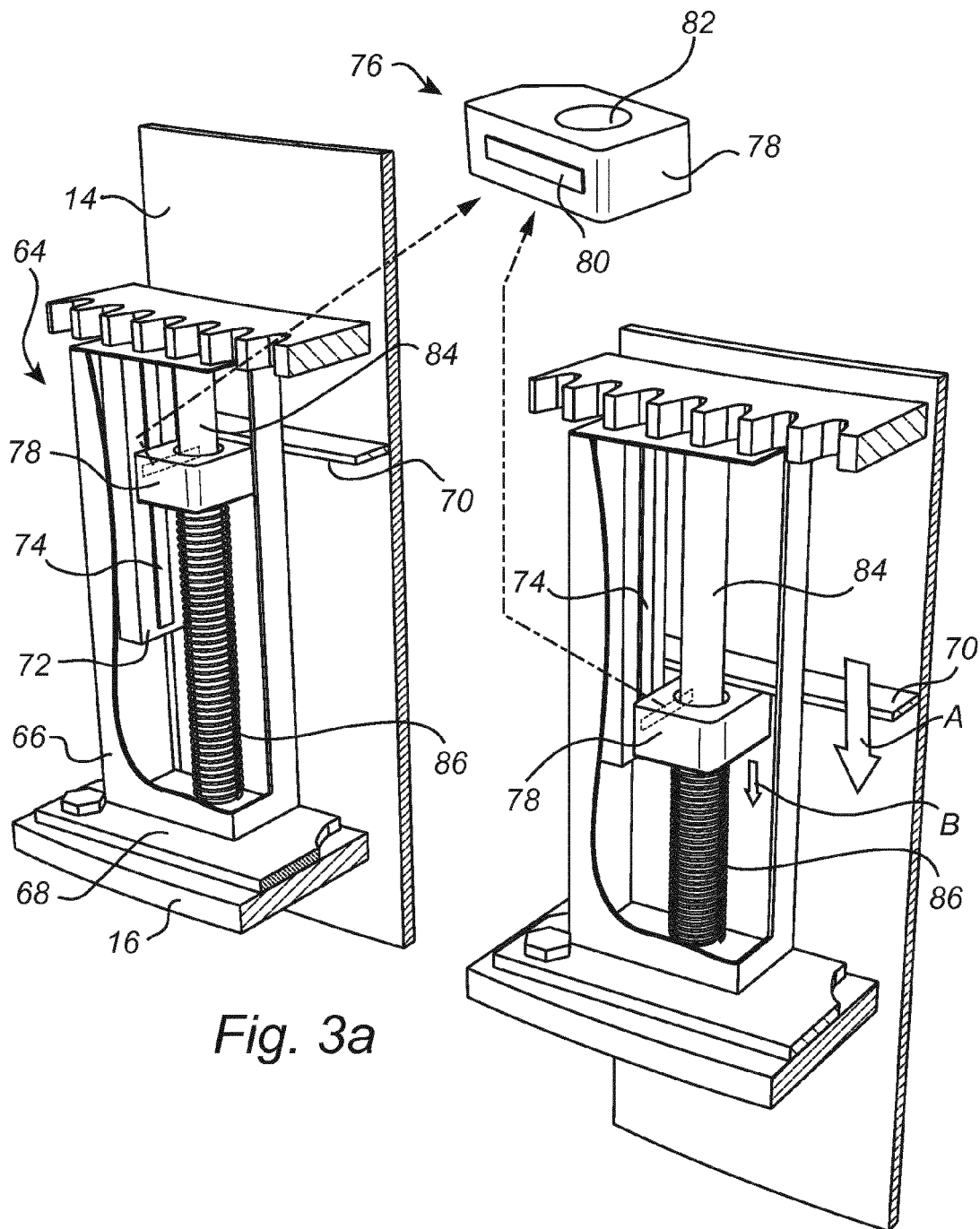


Fig. 3a

Fig. 3b

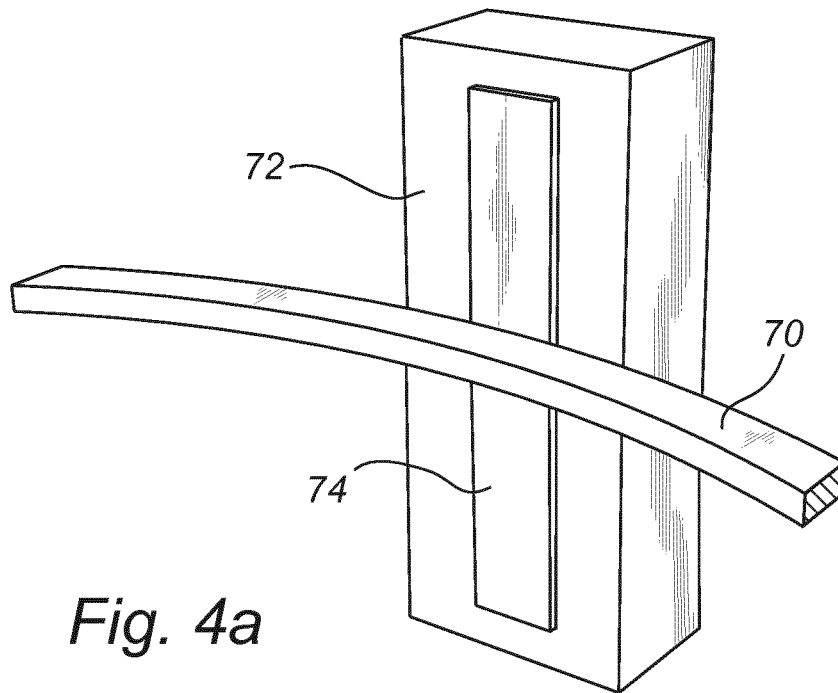


Fig. 4a

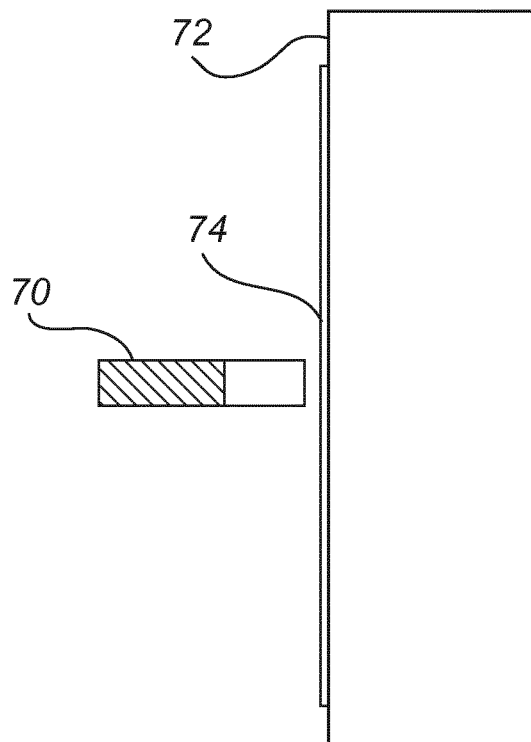


Fig. 4b

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CONE CRUSHER HAVING AN ARRANGEMENT FOR MEASURING A POSITION OF A CRUSHING SHELL

RELATED APPLICATION DATA

This application is a §371 National Stage Application of PCT International Application No. PCT/EP2012/072519 filed Nov. 13, 2012 claiming priority of EP Application No. 1191503.9, filed Dec. 1, 2011.

TECHNICAL FIELD OF THE INVENTION

The present invention relates to a cone crusher comprising an outer crushing shell and an inner crushing shell forming between them a crushing gap, the outer crushing shell being supported on an upper frame member in vertically adjustable engagement with a lower frame member, said vertically adjustable engagement being configured for adjusting the vertical position of the outer crushing shell relative to the lower frame member so as to permit adjustment of the width of the crushing gap. The cone crusher further comprises a sensor arrangement provided with a sensor element mounted to one of the lower frame member and the upper frame member.

BACKGROUND ART

A cone crusher may be utilized for efficient crushing of material, such as stone, ore etc., into smaller sizes. US 2010/0102152 A1 describes an exemplary cone crusher. In such a cone crusher, material is crushed between an outer crushing shell, which is mounted in a frame, and an inner crushing shell, which is mounted on a crushing head. The material is crushed by making the crushing head gyrate such that it rolls on the outer crushing shell via the material to be crushed.

The crusher of US 2010/0102152 A1 is provided with a proximity sensor for measuring the position of the outer crushing shell. The position of the outer crushing shell should be measured with a high accuracy to enable efficient crusher operation, and a crushed material having the desired properties.

SUMMARY OF THE INVENTION

An object of the present invention is to provide a crusher having an improved level of accuracy as regards the measurement of the vertical position of an adjustable crushing shell.

This object is achieved by a cone crusher comprising an outer crushing shell and an inner crushing shell forming between them a crushing gap, the outer crushing shell being supported on an upper frame member in vertically adjustable engagement with a lower frame member, said vertically adjustable engagement being configured for adjusting the vertical position of the outer crushing shell relative to the lower frame member so as to permit adjustment of the width of the crushing gap, and a sensor arrangement provided with a sensor element mounted to one of the lower frame member and the upper frame member for measuring the vertical position of the outer crushing shell, wherein the crusher further comprises a target means arranged to be detectable by said sensor element, wherein one of the target means and the sensor element being arranged to follow vertical movement of the upper frame member and to move in relation to the other one of the target means and the sensor element, said sensor element comprising a vertical sensing array which extends in a vertical direction along at least a portion of a

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range within which the target means may be moved upon adjusting the vertical position of the upper frame member, and the target means being detectable at different vertical positions along the vertical sensing array.

This crusher has the advantage that the vertical position of the outer crushing shell can be measured with high level of accuracy. This is possible since the distance between the sensor element and the target means may be short and well-defined along the entire range of movement of such a target. Hence, the sensor is arranged to look, e.g., for changes in an electro-magnetic sensing field, in a horizontal direction. By looking in a horizontal direction a constant distance between the target means and the sensor may be maintained.

The target means may comprise a circumferential flange in order to even further improve the level of accuracy in measuring the vertical position of the outer crushing shell and/or to enable reducing the risk of damage to sensitive component(s) of the sensor arrangement.

According to an embodiment the circumferential flange is externally arranged on the upper frame member and the sensor arrangement is mounted to the lower frame member to ensure measuring the vertical position in a robust and reliable manner.

Preferably the sensor element comprises a sensor that is able to detect the presence of a target means without any physical contact therewith. The sensor element and a target of the target means are preferably arranged with a clearance therebetween to prevent damage to the sensor element.

According to one embodiment the sensor element comprises an inductive sensor. The vertical sensing array may then comprise coils arranged in a coil array. An advantage of an inductive sensor is that such a sensor has a good ability to accurately measure the position of the target means in high shock and high vibration environments. This embodiment has the advantage that a very robust and reliable measuring of the vertical position is achieved.

The distance between the outer surface of said sensing array and a target of said target means may be substantially constant.

According to an embodiment the target means comprises a sliding member arranged to abut at least a portion of said flange. This embodiment has the advantage that the movement of a target of the target means, which target may be the sliding member itself or an element mounted on the sliding member, may be limited to vertical movements, which may improve the robustness and reliability of the sensor arrangement since possible damage to sensitive component(s) of the sensor arrangement is prevented.

The target means preferably comprises a resilient member arranged to press the sliding member against at least a portion of said flange in order to ensure that the sliding member follows vertical movements of the flange, and thereby movements of the upper frame member.

The resilient member is preferably a spring, such as a compression spring.

The sliding member is preferably provided with a target formed from a magnetic metallic material, such as steel, which is detectable by the sensor element. An advantage of this embodiment is that the sliding member itself does not need to be detectable by the sensor element and may thus be formed from a material that is chosen mainly with respect to suitable properties for sliding against the flange.

According to an embodiment the target of the target means has a vertical height which is in the range of 2 to 25 mm.

BRIEF DESCRIPTION OF THE DRAWINGS

The invention will hereafter be described in more detail and with reference to the appended drawings.

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FIG. 1 is a cross-section and illustrates, schematically, a cone crusher according to one embodiment.

FIG. 2 is a schematic perspective view of the cone crusher of FIG. 1 and illustrates a sensor arrangement according to a first embodiment.

FIG. 3a is a magnified view of the sensor arrangement illustrated in FIG. 2, illustrated in a first position.

FIG. 3b is a magnified view of the sensor arrangement, illustrated in a second position.

FIG. 4a is a schematic, partly sectional, perspective view and illustrates a sensor arrangement according to a second embodiment.

FIG. 4b is a schematic, partly sectional, side view and illustrates the sensor arrangement of FIG. 4a.

DETAILED DESCRIPTION OF PREFERRED EMBODIMENTS

FIGS. 1 and 2 illustrate a cone crusher 10 which is of the inertia cone crusher type. The cone crusher 10 comprises a crusher frame 12 in which the various parts of the crusher 10 are mounted. The frame 12 is suspended on cushions 11 to dampen vibrations occurring during the crushing action.

The crusher frame 12 comprises an upper frame member 14, which has the shape of a bowl, and a lower frame member 16. The upper frame member 14 is provided with an outer thread 18, which co-operates with an inner thread 20 of the lower frame member 16 in such a manner that the outer and inner threads 18, 20 together form a vertically adjustable engagement of the upper frame member 14 to the lower frame member 16 in the form of a threaded engagement 19.

The upper frame member 14 supports, on the inside thereof, an outer crushing shell 22. The lower frame member 16 supports an inner crushing shell arrangement 24. The inner crushing shell arrangement 24 comprises a crushing head 26, which has the shape of a cone and which supports an inner crushing shell 28. The outer and inner crushing shells 22, 28 form between them a crushing gap 30, to which material that is to be crushed is supplied.

The crushing head 26 rests on a spherical bearing 32, which is supported by the lower frame member 16. Hence, the crushing head 26 with the inner crushing shell 28 supported thereon, is supported by the lower frame member 16. The crushing head 26 is rotatably connected to an unbalance bushing 34, which has the shape of a cylindrical sleeve. An unbalance weight 36 is mounted on one side of the unbalance bushing 34. At its lower end the unbalance bushing 34 is connected to a drive shaft 38 via a transmission shaft 40. Universal joints 42 of the transmission shaft 40 allow the lower end of the unbalance bushing 34 to be displaced from a vertical axis A during operation of the crusher 10.

When the crusher 10 is in operation, the drive shaft 38 is rotated by a motor in a non-illustrated manner, e.g. via a belt-and-pulley transmission 43. The rotation of the drive shaft 38 causes the unbalance bushing 34 to rotate, and as an effect of that rotation the unbalance bushing 34 swings outwards in response to the centrifugal force to which the unbalance weight 36 is exposed. The combined rotation and swinging of the unbalance bushing 34 makes the crushing head 26 gyrate about a vertical axis, such that material is crushed in the crushing gap 30 formed between the outer and inner crushing shells 22, 28.

The width of the crushing gap 30 can be adjusted by turning the upper frame member 14, by means of the threads 18, 20, such that the vertical distance between the shells 22, 28 is adjusted. To this end, the upper frame member 14 is provided with a circumferential gear ring 44. The gear ring 44 is in

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mesh with a pinion 46, which is arranged to be rotated by a crushing gap adjustment motor (not shown) mounted within a motor bracket 62 fitted to the lower frame member 16. By operating the crushing gap adjustment motor, the pinion 46 turns the gear ring 44, and thereby also the upper frame member 14, such that the upper frame member 14 is vertically translated by the threaded engagement 19. Thereby, also the outer crushing shell 22 is vertically translated, such that the width of the crushing gap 30 is adjusted.

As is best illustrated in FIG. 2, the gear ring 44 is connected to the upper frame member 14 via keyed sliding engagements 57, which allow the gear ring 44 to remain in engagement with the pinion 46 while the upper frame member 14 is vertically translated. The keyed sliding engagements 57 are formed by vertical bars 56, attached to the upper frame member 14, which are keyed with mating notches 58 of the inner periphery of the gear ring 44. Thereby, the gear ring 44 is rotationally locked to the upper frame member 14, and may slide vertically along the bars 56. The gear ring 44 rests and slides, when being turned, upon an upper portion of the motor support bracket 62.

Continuing with the description of FIG. 2, the cone crusher 10 comprises a sensor arrangement 64 for measuring the vertical position of the upper frame member 14 and of the outer crushing shell 22 supported thereby. The sensor arrangement 64 comprises a sensor housing 66 which is mounted on a sensor housing bracket 68 fitted to the lower frame member 16.

The upper frame member 14 is provided with a circumferential flange 70, as will be described in more detail with reference to FIGS. 3a and 3b. The outwardly projecting flange 70 is fixedly mounted to the upper frame member 14 by bolts (not shown).

FIGS. 3a and 3b illustrate in greater detail the sensor arrangement 64. The sensor arrangement 64 comprises a sensor element 72 mounted within the sensor housing 66 on a side wall thereof. The sensor element 72 comprises an elongate vertical sensing array 74 which extends in a vertical direction. The vertical sensing array 74 may typically have a vertical height which is in the range of 50 to 2000 mm. The vertical sensing array 74 may typically have a horizontal width which is in the range of 0.1 to 200 mm. In this embodiment the sensor element 72 comprises an inductive position sensor. Such an inductive position sensor generates an inductive field which shifts along a sensitive surface and detects a metallic target in the detection range of the inductive field. The inductive sensor comprises several coils arranged in a coil array. Hence, in this embodiment the vertical sensing array 74 comprises a coil array. The inductive sensor calculates the current position of the target and provides output either as a distance-proportional analog signal or as a definable switching position. The inductive sensing field extends along the vertical height of the vertical sensing array 74. The sensor element 72 is thus able to detect the vertical position along the vertical sensing array 74 of a metal target without any physical contact therewith. The sensor output is received by a control unit (not shown) connected to the sensor element 72.

The crusher 10 comprises a target means 76 arranged to follow vertical movement of the upper frame member 14. In this embodiment the target means 76 comprises a sliding member 78, a metallic target 80 attached to the sliding member 78, and the circumferential flange 70.

The sliding member 78 is provided with a hole 82 in which a guiding rod 84 is received. The guiding rod 84 is mounted within the sensor housing 66 to guide movement of the sliding member 78. The sliding member 78 is arranged around the

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guiding rod **84** and is thus free to move in a vertical direction under the guidance of the guiding rod **84**. A compression spring **86** is arranged around the guiding rod **84** between the sliding member **78** and a bottom plate of the sensor housing **66** to exert a vertical pressing force on the sliding member **78**. The sliding member **78** is pressed vertically upwards and against the flange **70** and thus abuts a lower portion of the outwardly projecting flange **70**.

Vertical adjustment of the frame member **14** is achieved by rotating the gear ring **44** by means of pinion **46** and adjustment motor, as described hereinbefore with reference to FIG. **2**. In one example, starting out from the situation illustrated in FIG. **3a**, the upper frame member **14** is moved vertically downwards. Then the flange **70**, which is rigidly attached to the upper frame member **14**, is displaced downwards, as illustrated by arrow A in FIG. **3b**. The sliding member **78**, which abuts against a lower portion of the flange **70**, then follows the vertical movement of the flange **70**, as illustrated by arrow B in FIG. **3b**. Hence, movement of the sliding member **78** is in this case caused by the vertical movement of the upper frame member **14** via the flange **70**.

Upon adjustment of the upper frame member **14** in a direction upwards the flange **70** is displaced upwards. The sliding member **78**, which is pressed against a lower portion of the flange **70** by the compression spring **86**, then follows the vertical movement of the flange **70** due to the spring force of the spring **86**. Movement of the sliding member **78** upwards to follow the movement of the flange **70** upon an adjustment of the upper member **14** upwards is thus caused by the force exerted on the sliding member **78** by the compression spring **86**. Hence, the target means **76** comprising the sliding member **78** is arranged to follow vertical movement downwards as well as upwards of the upper frame member **14**.

In order to generate a sensor output the target means **76** is detectable by the inductive sensor of the sensor element **72** along the vertical sensing array **74**. To this end, the target means **76** comprises a metallic target **80**, which is attached to the sliding member **78** in such a manner that it moves with the sliding member **78**. The vertical sensing array **74** extends in a vertical direction along at least a portion of a vertical range within which the target **80** may be moved upon adjusting the vertical position of the upper frame member **14**. The metallic target **80** may have a vertical height which is in the range of, for example, 2 to 25 mm. The vertical height of the metallic target **80** is adapted to the type of sensor element **72** that is utilized. In accordance with one example, the vertical height of the metallic target **80** may be 13 mm. The vertical sensing array **74** of the sensor element **72** may detect the vertical position of the target **80** with a high accuracy. In accordance with one embodiment, the sliding member **78** is made from an isolating material, such as a plastic material, in order not to interfere with the desired electrical interference between the target **80** and the vertical sensing array **74** of the sensor element **72**.

The guiding rod **84** is arranged such that a constant clearance of typically about 0 to 20 mm, for example a clearance of 5 mm, is formed between the metallic target **80** and the outer surface of the sensing array **74**. Hence, the horizontal distance between the target **80** and the sensing array **74** is substantially constant, regardless of the actual vertical position of the upper frame member **14**, which provides for measuring the position of the upper frame member **14** with a high level of accuracy.

During operation of the sensor arrangement **64**, the sensor element **72** emits an alternating electro-magnetic sensing field along the sensing array **74**. When the metallic target **80** enters the sensing field eddy currents are induced in the target **80** which reduces the signal amplitude of the sensor element

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72 and triggers a change of state at the sensor element **72** output received by the control unit. In this embodiment an inductive sensor called PMI-F110 commercially available from Pepperl+Fuchs GmbH was used.

Hereinafter a crusher according to a second embodiment will be described with reference to FIGS. **4a** and **4b**. Many features disclosed in the first embodiment are also present in the second embodiment with similar reference numerals identifying similar or same features. Having mentioned this, the description will focus on explaining the differing features of the second embodiment.

In the second embodiment the target of the target means is formed by the circumferential flange **70** itself. The sensor arrangement comprises an inductive sensor element **72** provided with a vertical sensing array **74** which extends in vertical direction along at least a portion of a vertical range within which the target means, i.e. the flange **70**, may be moved upon adjusting the vertical position of the upper frame member **14**. The sensor element **72** is arranged such that the sensing array **74** faces a front edge of the flange **70**. The flange **70** is thus detectable at different vertical positions along the vertical sensing array **74**. The sensor element **72** is arranged such that a clearance is formed between the front edge of the flange **70** and the sensing array **74**, as best illustrated in FIG. **4b**. In this embodiment a sliding member as disclosed in the first embodiment is not needed since the flange **70** itself is detectable by the sensing array **74**.

The invention has mainly been described above with reference to a few embodiments. However, as is readily appreciated by a person skilled in the art, other embodiments than the ones disclosed above are equally possible within the scope of the invention, as defined by the appended patent claims.

For example, the invention is not limited to any particular type of cone crusher; on the contrary, it is suited for many different types of cone crushers known to those skilled in the art, such as the type of crusher having the top of a head shaft journaled in a spider assembly, as well as the type of crusher that is described in U.S. Pat. No. 1,894,601, occasionally called Symons type, and the inertia type of cone crushers disclosed herein, having an unbalance weight for obtaining a gyratory motion of the crushing head.

The sensor element may comprise another type of sensor than the one described hereinbefore. For instance, the sensor element may comprise a capacitive or photoelectric sensor. In case the sensor element comprises a photoelectric sensor the target means preferably comprises a plastic target.

Hereinbefore it has been described that the sensor element **72** is attached to the lower frame member **16** and the target means is arranged to follow the vertical movement of the upper frame member **14**. In an alternative embodiment a sensor element may instead be arranged to follow the vertical movement of the upper frame member **14**, e.g. by way of abutment against the flange **70**. A stationary target, e.g. in the form of a circumferential rim, may then be fixedly arranged to the lower frame member **16**.

Instead of a compression spring **86** as disclosed in the first embodiment another arrangement may be used to press the sliding member **78** against at least a portion of the flange **70**. For instance, a tension spring may be used for this purpose. Then the sliding member **78** is arranged to abut at least an upper portion of the flange.

Hereinbefore it has been described that the circumferential flange **70** is fixedly secured to the upper frame member **14**. It is realised that a circumferential flange may, as alternative, be integrally formed with the upper frame member **14**.

Further objects and features of the present invention will be apparent from the following detailed description and claims.

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The invention claimed is:

1. A cone crusher comprising:
an outer crushing shell and an inner crushing shell forming
between them a crushing gap, the outer crushing shell
being supported on an upper frame member in vertically
adjustable engagement with a lower frame member, said
vertically adjustable engagement being configured for
adjusting the vertical position of the outer crushing shell
relative to the lower frame member so as to permit
adjustment of a width of the crushing gap;
a sensor arrangement provided with a sensor element, the
sensor arrangement being mounted to the lower frame
member for measuring the vertical position of the outer
crushing shell; and
a target means arranged to be detectable by said sensor
element, the target means being arranged to move ver-
tically with the upper frame member and to move in
relation to the sensor element, said sensor element
including a vertical sensing array which extends in a
vertical direction along at least a portion of a range
within which the target means may be moved upon
adjusting the vertical position of the upper frame mem-
ber, and the target means being detectable at different
vertical positions along the vertical sensing array.
2. A cone crusher according to claim 1, wherein the target
means is a circumferential flange.
3. A cone crusher according to claim 2, wherein said cir-
cumferential flange is externally arranged on the upper frame
member and said sensor arrangement is mounted to the lower
frame member.

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4. A cone crusher according to claim 1, wherein the target
means includes a target and the sensing array and the target of
the target means are arranged with a clearance therebetween.
5. A cone crusher according claim 1, wherein the sensor
element includes an inductive sensor.
6. A cone crusher according claim 1, wherein a distance
between an outer surface of said sensing array and a target of
said target means is constant.
7. A cone crusher according to claim 2, wherein the target
means includes a sliding member arranged to abut at least a
portion of said flange.
8. A cone crusher according to claim 7, wherein the target
means includes a resilient member arranged to press the slid-
ing member against at least a portion of said flange.
9. A cone crusher according to claim 8, wherein said resil-
ient member is a compression spring.
10. A cone crusher according to claim 7, wherein said
sliding member is provided with a target which is detectable
by the sensor element.
11. A cone crusher according to claim 1, wherein said
target means includes a target formed from a magnetic metal-
lic material.
12. A cone crusher according to claim 11, wherein the
target has a vertical height which is in the range of 2 to 25 mm.
13. A cone crusher according to claim 11, wherein the
magnetic metallic material is steel.

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